



## Secondary flows of viscoelastic fluids in serpentine microchannels

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### Abstract

Secondary flows are ubiquitous in channel flows, where small velocity components perpendicular to the main velocity appear due to the complexity of the channel geometry and/or that of the flow itself such as from inertial or non-Newtonian effects. We investigate here the inertialess secondary flow of viscoelastic fluids in curved microchannels of rectangular cross-section and constant but alternating curvature: the so-called “serpentine channel” geometry. Numerical calculations (Poole et al. *J Non-Newton Fluid Mech* 201:10–16, 2013) have shown that in this geometry, in the absence of elastic instabilities, a steady secondary flow develops that takes the shape of two counter-rotating vortices in the plane of the channel cross-section. We present the first experimental visualization evidence and characterization of these steady secondary flows, using a complementarity of  $\mu$ PIV in the plane of the channel, and confocal visualisation of dye-stream transport in the cross-sectional plane. We show that the measured streamlines and the relative velocity magnitude of the secondary flows are in qualitative agreement with the numerical results. In addition to our techniques being broadly applicable to the characterisation of three-dimensional flow structures in microchannels, our results are important for understanding the onset of instability in serpentine viscoelastic flows.

**Keywords** Polymer solutions · Non-Newtonian fluids · Vortices · Confocal microscopy · Particle image velocimetry

### 1 Introduction

Three-dimensional velocity fields are widespread in channel and pipe flows, where the geometry of the duct can combine with the properties of the base primary flow (i.e. the flow in the streamwise direction) to trigger a weak current with velocity components perpendicular to the streamwise direction. Therefore, the ability to measure, and understand, the velocity field in all three directions of such flows is of general importance. In microfluidic flows, however, the flow-field in the streamwise direction is often the only component characterised, because of optical access limitations and due to the fact that the absolute value of the other velocity components are typically very small (Tabeling 2005). Despite their small magnitude, such secondary flows are often ultimately responsible for enhanced mixing (above that due to diffusion alone) of mass and heat which is a frequent aim of various microfluidic devices (Lee et al. 2011; Mitchell 2001; Stroock et al. 2002; Amini et al. 2013; Hardt et al. 2005; Kockmann et al. 2003). Secondary flows also have important implications in particle focussing (Del Giudice et al. 2015;

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Di Carlo et al. 2007) where they may either be exploited or act as a hindrance.

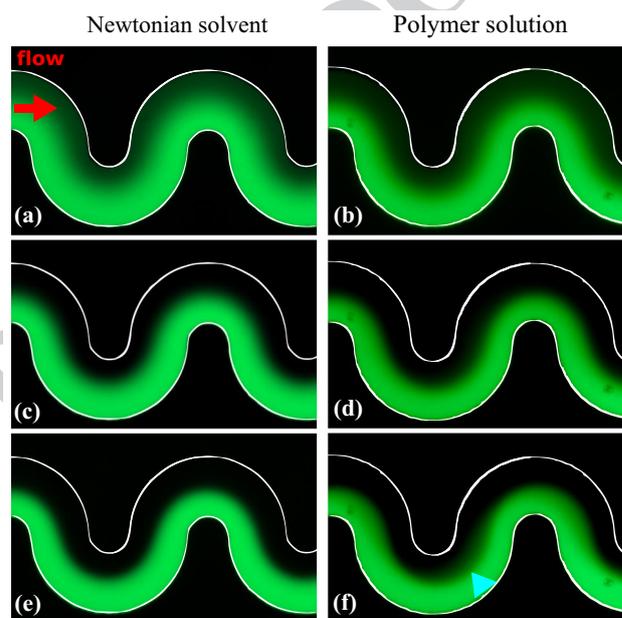
Secondary flows in microfluidic systems may be driven by the complexity of the channel geometry only: for the creeping flow of a Newtonian fluid, Lauga et al. (2004) have shown that a secondary flow must develop if the channel has both varying cross-section and streamwise curvature. Changes in the streamwise curvature of channels with constant cross-section have also been shown to give rise to a secondary flow around the bend (Guglielmini et al. 2011; Sznitman et al. 2012). More complex geometries have been designed to obtain chaotic micromixers, in which secondary flows are triggered and expose volumes of fluid to a repeated sequence of rotational and extensional local flows (Ottino 1989; Stroock et al. 2002; Amini et al. 2013).

Complexity in the equations of fluid motion is another driving mechanism for secondary flows: although usually not dominant at the microscale, inertia can play a role in microfluidic systems (Di Carlo 2009; Amini et al. 2014). Combined with the flow geometry, it drives secondary flows such as the well-known “Dean” vortices (Dean 1927, 1928) observed in curved channels and pipes, or the steady vortical structure of the “engulfment” regime in T-junction mixers (Kockmann et al. 2003; Fani et al. 2013). In the absence of inertia, viscoelasticity is another source of fluid dynamic complexity: viscoelastic analogues of the Dean vortices are formed, in the creeping-flow regime, by the coupling of the first normal-stress difference with streamline curvature (Robertson and Muller 1996; Fan et al. 2001; Poole et al. 2013; Bohr et al. 2018). Note that second-normal stress differences in viscoelastic fluids may also drive an inertia-less secondary motion in ducts of non-axisymmetric cross-section, but this flow is typically much weaker (Gervang and Larsen 1991; Debbaut et al. 1997; Xue et al. 1995). We emphasise that in listing those potential sources of secondary flow we are not attempting to be exhaustive, but simply to illustrate that they may occur under many different scenarios.

We focus here on the viscoelastic secondary flow driven by streamline curvature. This steady secondary flow is always present in the steady flow of viscoelastic fluids in curved geometries, and pertains at all flow rates until a critical flow rate is reached at which the flow becomes time-dependent due to a well-known purely elastic instability (Groisman and Steinberg 2000; Arratia et al. 2006; Afik and Steinberg 2017; Souliès et al. 2017). Characterising this secondary flow is thus essential to the knowledge of the three-dimensional base flow from which the elastic instability develops: its structure may interact with the onset of the instability, as hypothesised to explain the partially unaccounted for stabilisation of shear-thinning viscoelastic flow in curved microchannels (Casanellas et al. 2016). It is also important for mixing and particle focusing applications that

rely on viscoelastic fluids (Groisman and Steinberg 2000; Del Giudice et al. 2015).

Evidence for such secondary flows is readily observable in simple visualisation experiments. By way of example, in Fig. 1 we show a classical experiment for the visualisation of mixing efficiency in a serpentine microchannel (Groisman and Steinberg 2000): two streams of the same fluid, one of them dyed with fluorescein, are co-injected into the serpentine micromixer. When a Newtonian fluid is injected, mixing is achieved by diffusion alone, which broadens the interface. With increasing flow rate, the residence time decreases, and so does the width of the interface. When a



**Fig. 1** Visualisation of mixing in a serpentine microchannel (the channel edges are highlighted in white): two streams of fluid are co-injected in a Y-junction, one of them being fluorescently labelled. Data for a viscoelastic polymer solution are displayed on the right-hand side, while data for the Newtonian solvent (a mixture of water and glycerol at 75–25 wt%) (The small fluorescein molecule diffuses almost freely in the polymer network and thus probes a local viscosity that is lower than the shear viscosity of the polymer solution as measured with a rheometer. The same behaviour has been quantified in solutions of (hydroxypropyl) cellulose (Mustafa et al. 1993), dextran (Furukawa et al. 1991) and polyethylene glycol (Holyst et al. 2009). For this reason, we use the solvent of the polymer solution as a Newtonian reference fluid. The slightly lower diffusion in the polymeric solution shows that the contribution from the polymer to the local viscosity, albeit small, is not entirely negligible.) are shown on the left-hand side. The flow rate increases from 2  $\mu\text{l}/\text{min}$  (top row) to 6  $\mu\text{l}/\text{min}$  (middle row) and 12  $\mu\text{l}/\text{min}$  (bottom row). At low flow rates (a, b) the interface between the two streams is broadened in both cases by the strong diffusion of the dye. At larger flow rates (c and d), the interface sharpens all along the channel due to the decreasing residence time. Further increase of the flow rate (e and f) leads to further sharpening of the interface for the Newtonian flow (e), but an additional spatially varying “blur” develops in the viscoelastic flow (f), blue triangular arrow

viscoelastic fluid is used, the evolution of the width of the interface with increasing flow rate is very different. At small flow rates a very broad interface is again observed, which initially sharpens when the flow rate is increased. When the flow rate is further increased, however, the interface locally widens again. This effect cannot be attributed to diffusion, which becomes less important with increasing flow rate (thus decreasing residence times). In addition, an asymmetry can be observed with the interface being significantly wider towards the end of each loop and a sharpening of the interface at the beginning of each new loop. This observation can only be explained with an underlying three dimensional flow structure that reverses direction in between consecutive loops. Previous numerical simulations (Poole et al. 2013) have shown the occurrence of a steady secondary flow in this serpentine channel geometry for dilute viscoelastic liquids. One of the aims of this work is to demonstrate and quantify the occurrence of this secondary flow experimentally by direct measurement. More generally, we show how different experimental methods can be used to determine quantitative information of generic secondary flows in micro-devices.

Being typically very weak (on the order of a few percent of the bulk primary velocity), secondary flows are hard to resolve even in macro-sized classical fluid mechanics experiments (Gervang and Larsen 1991). Thus it is not surprising that such flows have been little characterised at the microscale. A number of recent experimental approaches may alleviate this issue, in particular the holographic microparticle tracking velocimetry ( $\mu$ PTV) technique (Salipante et al. 2017), confocal microparticle image velocimetry (confocal  $\mu$ PIV) (Li et al. 2016) or using standard particle image velocimetry in conjunction with a channel design and material that allow for microscope observation in several perpendicular planes (Burshtein et al. 2017).

Here, we will characterise experimentally the three-dimensional structure of the flow with supporting numerical simulations that match the geometrical conditions. Our aim is to use a complementarity of  $\mu$ PIV, confocal microscopy and insight gleaned from simulation to quantify the secondary flow and confirm its vortical structure and sense of rotation. Our techniques are very generic and thus broadly applicable to the characterisation of three-dimensional flow structures in microchannels.

## 2 Experimental and numerical methods

### 2.1 Working fluids and rheological characterisation

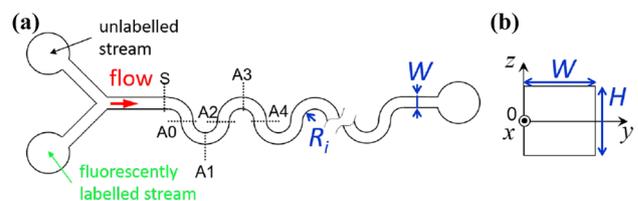
Model viscoelastic fluids were prepared by dissolving polyethylene oxide (PEO, from Sigma Aldrich) with a molecular weight of  $M_w = 4 \times 10^6$  g/mol in a water/glycerol (75–25% in weight) solution. The PEO was supplied from the same

batch as used in Casanellas et al. (2016). The solvent viscosity at  $T = 21$  °C is  $\eta_s = 2.1$  mPa s (data not shown). The polymer concentration was fixed to  $c = 500$  ppm (w/w). The total viscosity of the resulting solution at  $T = 21$  °C is  $\eta = 3.8$  mPa s giving a solvent-to-total viscosity ratio  $\beta = 0.55$ . The overlap concentration for this polymer in water is  $c^* \approx 550$  ppm (Casanellas et al. 2016). Although this solution is close to the semi-dilute limit, we confirmed that shear-thinning effects, of both the shear viscosity and the first normal-stress difference, are essentially negligible [see e.g. Casanellas et al. (2016)].

### 2.2 Microfluidic geometry

We tested the polymer solution in serpentine microchannels consisting of nine half loops. A sketch of the channel is shown in Fig. 2. We note that, in this channel geometry, the absolute value of the curvature is constant along the channel but the sign of the curvature changes from positive to negative between consecutive half-loops. This change of sign is not required for the development of the viscoelastic secondary flow, but is a feature of our two-dimensional geometry which conveniently allows for the study of several consecutive loops at a constant radius of curvature. Numerical simulations of the creeping flow of Newtonian fluids in bent microchannels show that this change in curvature is expected to trigger a local secondary flow where subsequent half-loops reconnect, but this flow quickly decays in the regions of constant curvature (Guglielmini et al. 2011), where our velocity measurements were made. Most of our results were obtained on a channel of nearly square cross-section, with a width  $W = 110 \pm 3$   $\mu$ m, height  $H = 99 \pm 1$   $\mu$ m and an inner radius of curvature (measured at the inner wall of the channel)  $R_i = 40 \pm 1$   $\mu$ m. Additional channels of comparable cross-sectional dimensions but larger radii of curvature were used for comparison.

The microchannels were fabricated in polydimethylsiloxane (PDMS), using standard soft-lithography



**Fig. 2** Schematic of the microfluidic geometry used: **a** top view and **b** cross-sectional view displaying the choice of axes.  $x$  is the primary flow velocity direction,  $y$  is the wall-normal direction (where the origin is taken at the inner edge of each loop), and  $z$  is the spanwise (vertical) direction. Therefore, the  $x$ ,  $y$ ,  $z$  coordinate system we consider is not fixed in space but advected with the flow. The location of the dyed stream used for confocal visualisation is also indicated

191 microfabrication methods (Tabeling 2005), and mounted  
 192 on a glass coverslip. The fluid was injected into the channel  
 193 via two inlets using two glass syringes (Hamilton, 500  $\mu\text{l}$   
 194 each) that were connected to a high-precision syringe pump  
 195 (Nemesys, from Cetoni GmbH). The experimental protocol  
 196 consisted of stepped ramps of increasing flow rate from 2  $\mu\text{l}$   
 197 /min up to a maximum of 20  $\mu\text{l}/\text{min}$ , with a flow rate step  
 198 of 2  $\mu\text{l}/\text{min}$ . The resolution of the applied flow rate was controlled  
 199 at a precision of  $\pm 0.2 \mu\text{l}/\text{min}$ , as confirmed independently  
 200 using a flow sensor (Flow unit S, from Fluigent, at  
 201 low flow rates and SLI-0430 Liquid flow meter, from Sen-  
 202 siorion for  $Q \geq 6 \mu\text{l}/\text{min}$ ). The step duration was set to 120 s,  
 203 and the measurements performed over the last 60 s, which  
 204 we confirm was long enough to ensure flow steadiness and  
 205 the decay of any initial transient regime. Experiments were  
 206 continued until the onset of the purely-elastic instability  
 207 where the flow became time-dependent. For the  $R_i = 40 \mu\text{m}$   
 208 channel this occurred at 14  $\mu\text{l}/\text{min}$  which we define as  $Q_{\text{insta}}$ .  
 209 At the onset of flow instability the Reynolds number  
 210 ( $Re = \rho U W / \eta$ , where  $\rho$  is the fluid density and  $U$  the mean  
 211 velocity) is small in all experiments, being at most 0.6.  
 212 Therefore, in our microfluidic flow experiments inertial  
 213 effects can be disregarded.

214 **2.3 Micro-particle image velocimetry**

215 Quantitative two-dimensional measurements of the flow field  
 216 were made in the  $xy$ -centreplane ( $z = 0$  plane) of the serpen-  
 217 tine device (Fig. 2) using a micro-particle image velocimetry  
 218 ( $\mu\text{PIV}$ ) system (TSI Inc., MN) (Meinhart et al. 2000; Were-  
 219 ley and Meinhart 2005). For this purpose, no fluorescent  
 220 dye was used but the test fluid was seeded with 0.02 wt%  
 221 fluorescent particles (Fluoro-Max, red fluorescent micro-  
 222 spheres, Thermo Scientific) of diameter  $d_p = 0.52 \mu\text{m}$  with  
 223 peak excitation and emission wavelengths of 542 nm and  
 224 612 nm, respectively. The microfluidic device was mounted  
 225 on the stage of an inverted microscope (Nikon Eclipse Ti),  
 226 equipped with a 20 $\times$  magnification lens (Nikon, NA = 0.45).  
 227 With this combination of particle size and objective lens, the  
 228 measurement depth over which particles contribute to the  
 229 determination of the velocity field was  $\delta z_m \approx 13 \mu\text{m}$  (Mein-  
 230 hart et al. 2000), which is approximately 13% of the channel  
 231 depth.

232 The  $\mu\text{PIV}$  system was equipped with a 1280  $\times$  800  
 233 pixel high speed CMOS camera (Phantom MIRO, Vision  
 234 Research), which operated in frame-straddling mode and  
 235 was synchronized with a dual-pulsed Nd:YLF laser light  
 236 source with a wavelength of 527 nm (Terra PIV, Continuum  
 237 Inc., CA). The laser illuminated the fluid with pulses of  
 238 duration  $\delta t < 10$  ns, thus exciting the fluorescent particles,  
 239 which emitted at a longer wavelength. Reflected laser light  
 240 was filtered out by a G-2A epifluorescent filter so that only  
 241 the light emitted by the fluorescent particles was detected

by the CMOS imaging sensor array. Images were captured  
 in pairs (one image for each laser pulse), where the time  
 between pulses  $\Delta t$  was set such that the average particle dis-  
 placement between the two images in each pair was around  
 4 pixels. Insight 4G software (TSI Inc.) was used to cross-  
 correlate image pairs using a standard  $\mu\text{PIV}$  algorithm and  
 recursive Nyquist criterion. The final interrogation area of  
 16  $\times$  16 pixels provided velocity vector spaced on a square  
 grid of 6.4  $\times$  6.4  $\mu\text{m}$  in  $x$  and  $y$ . The velocity vector fields  
 were ensemble-averaged over 50 image pairs.

252 **2.4 Confocal microscopy**

Vertical images of the cross-section of the channel (in  $yz$   
 planes) were obtained by confocal microscopy imaging  
 using dyed stream visualisation (as illustrated in the  $xy$  plane  
 in Fig. 1).  $z$ -stacks of two-dimensional images of size 1024 $\times$   
 1024 pixels in  $xy$  planes were acquired at a rate of 6 fps  
 using a laser line-scanning confocal fluorescence micro-  
 scope (LSM 5 Live, Zeiss), with a 40 $\times$  water immersion  
 objective lens (1.20 NA). The voxel size was 0.16  $\times$  0.16  $\times$   
 0.45  $\mu\text{m}$  in the  $x - y - z$  direction.

262 **2.5 Viscoelastic constitutive equation, numerical**  
 263 **method and structure of predicted secondary**  
 264 **flow**

265 The three-dimensional numerical simulations assume  
 266 isothermal flow of an incompressible viscoelastic fluid  
 267 described by the upper-convected Maxwell (UCM)  
 268 model (Oldroyd 1950) in a channel of matched dimensions  
 269 to those used in the experiments. The equations that need to  
 270 be solved are those of mass conservation,

$$\nabla \cdot \mathbf{u} = 0, \tag{1}$$

and the momentum balance,

$$-\nabla p + \nabla \cdot \boldsymbol{\tau} = \mathbf{0}, \tag{2}$$

274 assuming creeping-flow conditions (i.e. the inertial terms are  
 275 exactly zero), where  $\mathbf{u}$  is the velocity vector with Cartesian  
 276 components ( $u_x, u_y, u_z$ ), and  $p$  is the pressure. For the UCM  
 277 model the evolution equation for the polymeric extra-stress  
 278 tensor,  $\boldsymbol{\tau}$ , is

$$\boldsymbol{\tau} + \lambda \boldsymbol{\tau}_{(1)} = \eta \dot{\boldsymbol{\gamma}}, \tag{3}$$

280 where  $\boldsymbol{\tau}_{(1)}$  represents the upper-convected derivative of  $\boldsymbol{\tau}$  and  
 281  $\eta$  the constant polymeric contribution to the viscosity of the  
 282 fluid, respectively.

283 Although the UCM model exhibits an unbounded steady-  
 284 state extensional viscosity above a critical strain rate ( $1/2\lambda$ ),  
 285 in shear-dominated serpentine channel geometries such  
 286 model deficiencies are unimportant and it is arguably the  
 287 simplest differential constitutive equation which can capture

288 many aspects of highly-elastic flows. Many more complex  
 289 models [e.g. the FENE-P, Giesekus and Phan–Thien–Tanner  
 290 models, see e.g. Bird et al. (1987)], simplify to the UCM  
 291 model in certain parameter limits and thus its general-  
 292 ity makes it an ideal candidate for fundamental studies of  
 293 viscoelastic fluid flow behaviour. The governing equations  
 294 were solved using a finite-volume numerical method, based  
 295 on the logarithm transformation of the conformation ten-  
 296 sor. Additional details about the numerical method can be  
 297 found in Afonso et al. (2009, 2011) and in other previous  
 298 studies [e.g. Alves et al. (2003a, b)]. For low Deborah  
 299 numbers  $De = \lambda U/R_i$  (with  $\lambda$  the relaxation time of the fluid,  $U$   
 300 the mean velocity and  $R_i$  the inner radius of curvature), the  
 301 numerical solution converges to a steady solution, which  
 302 was assumed to occur when the  $L^1$  norm of the residuals of  
 303 all variables reached a tolerance of  $10^{-6}$ . Beyond a critical  
 304 Deborah number, a time-dependent purely-elastic instabil-  
 305 ity occurs. The numerical results in the current paper are  
 306 restricted to Deborah numbers below the occurrence of this  
 307 purely-elastic instability, thus the flow remains steady in all  
 308 simulations.

309 The structure and strength of viscoelastic secondary flows  
 310 in a serpentine geometry have been investigated in detail  
 311 by Poole et al. (2013). The main results of this study are  
 312 recalled below, and lay the foundations for the simulations  
 313 that we carry out here, which are focused on the geometry of  
 314 the experimental system we used. The projected streamlines  
 315 in the  $yz$  plane of the computed secondary flow are shown  
 316 in Fig. 3a. The flow takes the shape of a pair of counter-  
 317 rotating vortices in the cross-sectional plane of the channel.  
 318 It is driven by the hoop stress, which drives the fluid towards  
 319 the inner side of the bend close to the top and bottom walls  
 320 (where the shear rate and thus the first normal stress differ-  
 321 ence are larger, as illustrated in Fig. 3b). The fluid is then  
 322 carried back towards the outer edge of the bend at the centre-  
 323 plane ( $z = 0$ ). Although the driving mechanism is dif-  
 324 ferent, the resulting qualitative features of this elasticity-  
 325 driven secondary flow are thus similar to the inertia-driven  
 326 Dean vortices (Dean 1928). The strength of the viscoelastic  
 327 secondary flow increases with the elastic contribution to  
 328 the flow (increasing  $De$ ) and the curvature of the channel.  
 329 Poole et al. (2013) have shown that far from the onset of  
 330 the purely elastic instabilities, the magnitude of the second-  
 331 ary flow, as quantified by the maximum spanwise velocity  
 332  $u_{z,max}$ , scales linearly with  $De$  (see Fig. 3c). The structure  
 333 of the flow has been shown to remain identical for aspect  
 334 ratios  $W/H$  varying from 1 up to 4; accordingly, the scaling  
 335 for  $u_{z,max}$  with  $De$  is not modified by the aspect ratio of  
 336 the channel. The scaling of the secondary flow strength with  
 337 the solvent viscosity contribution has also been assessed (Poole  
 338 et al. 2013), and can be expressed as an effective Deborah  
 339 number  $De_{eff} = (1 - \beta)De$  where  $\beta$  is the ratio of the solvent

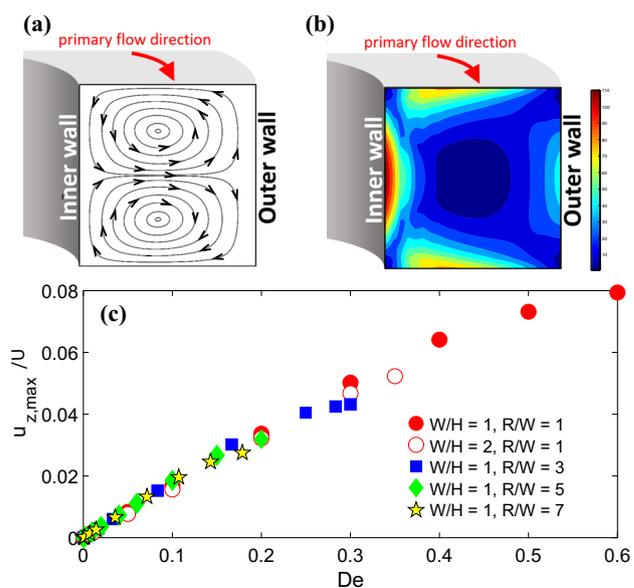


Fig. 3 a Numerically predicted structure of the secondary flow: two counter-rotating vortices develop in the cross-sectional plane of the channel. b Contour plot of the first normal stress difference ( $N_1/(\eta U/W)$ ), in the cross-section of the channel ( $De = 0.5$ ,  $W/H = W/R = 1$ ): the strong positive and asymmetric normal stress difference at the top and bottom wall drives the secondary flow. c Numerically calculated scaling of the strength of the secondary flow: the maximum spanwise velocity scales linearly with  $De$  over a wide range of aspect ratio and radius of curvature parameters [adapted from Poole et al. (2013)]

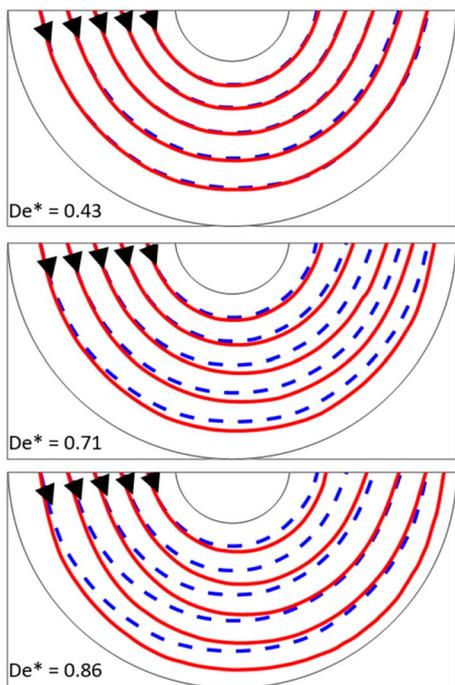
viscosity to the total (solvent + polymeric) viscosity. In the UCM model,  $De_{eff} = De$ .

The precise experimental determination of the relaxation time of the fluid is difficult for dilute polymer solutions so that the determination of  $De$  for our experimental data is challenging. However, we can use the onset of the purely elastic instability as a reference point to match the Deborah numbers in our numerical and experimental data. For a given serpentine geometry, the flow becomes unstable beyond a critical flow speed, usually expressed in terms of a critical Weissenberg number to quantify the importance of the elastic contribution to the flow:  $Wi_{insta}(R_i) = \lambda U_{insta}/W$  (with  $Wi = De \times R_i/W$  the Weissenberg number) (Zilz et al. 2012). For a given channel geometry  $Wi/Wi_{insta} = De/De_{insta} = Q/Q_{insta}$ . Therefore, to enable quantitative comparison between experimental and numerical results, all the results for the channel geometry described above will be presented in terms of the reduced quantity  $De^* = De/De_{insta} = Q/Q_{insta}$ , with  $De_{insta} \approx 1.24$  from the numerical results for the experimental geometry we used ( $R_i/W = 0.36$ ).  $De^*$  is independent of  $\beta$  and, therefore, the solvent viscosity does not need to be taken into account in the present numerical simulations.

### 3 Results

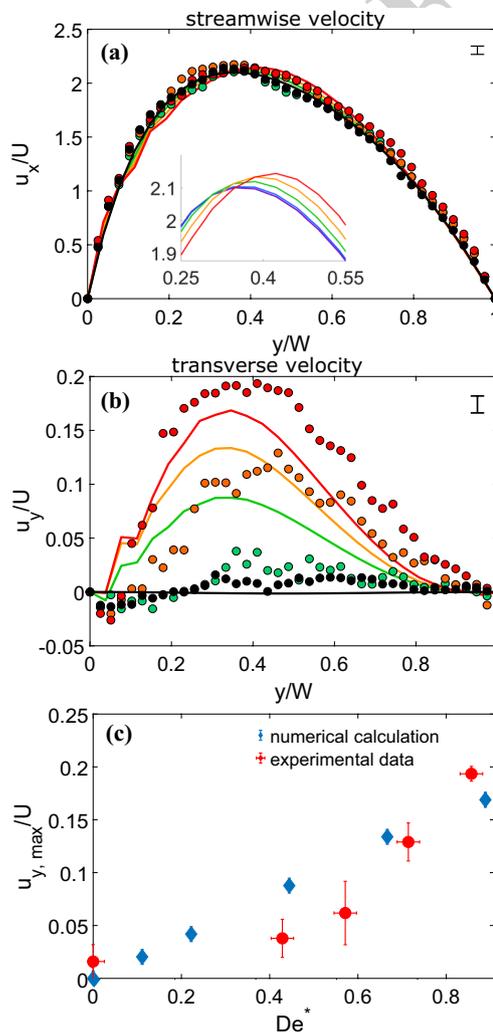
#### 3.1 Flow measurement in the plane of the channel using $\mu$ PIV

In this section, we show that classical  $\mu$ PIV measurements in the  $xy$  centreplane provide significant evidence of the existence of a secondary flow. In this symmetry plane,  $u_z = 0$  so that the velocity field is fully characterised by the 2D PIV. In Fig. 4 we show streamlines constructed from the measured velocity field along the first half loop (i.e. from A0 to A2 as shown in Fig. 2). The blue dashed line highlights the Newtonian result which can be seen to travel in approximate concentric semicircles around the bend. The red lines indicate the streamlines of the polymeric solution, which, at low Deborah number, can be seen to match the Newtonian ones very closely. In contrast, with increasing  $De^*$  (increasing flow rate) there is a marked deviation of the streamlines for the viscoelastic fluid away from the inner bend towards the outer, in good agreement with the sense of the secondary flow predicted by the numerical simulations (Poole et al. 2013) and as discussed above and shown in Fig. 3. These streamline patterns thus provide our first piece of qualitative evidence for the existence of an elastic secondary flow.



**Fig. 4** Red lines: experimental flow streamlines in the channel centreplane, for increasing values of flow elasticity. For comparison, the streamlines for a Newtonian solution of the same viscosity are shown with a blue dashed line: a clear deviation of the streamlines towards the outer edge of the bend occurs in viscoelastic flow

To support these qualitative streamline observations in a more quantitative sense, in Fig. 5 we plot the velocity components around location A1. The primary and wall-normal components of both the experimentally determined (symbols) and the numerically computed (full lines) velocity fields have been averaged over an angular sector of  $10^\circ$  upstream and downstream of A1.  $y$  describes as usual the wall-normal direction, and  $x$  the primary velocity direction: the primary velocity component is  $u_x$



**Fig. 5** Experimental (bullets) and numerical (full lines) velocity plots at location A1 (see Fig. 2) for the primary (a) and wall-normal velocity (b), for increasing values of  $De^*$  (black bullet Newtonian fluid, green bullet 0.43, orange bullet 0.71, red bullet 0.86 and black full line 0, pink full line 0.11, blue full line 0.22, green full line 0.44, orange full line 0.67, red full line 0.89). For easier comparison, only select  $De^*$  values are shown in b. The zero  $y$  location is taken at the inner edge of the bend, with the geometry being that described in Sect. 2.2:  $R_i/W = 0.36$ . The peak streamwise velocity shifts with  $De$ , as is more visible in the inset of a. The scale bar in the top right corner indicates the magnitude of the error on the experimental data. c Scaling for the maximum wall-normal velocity with  $De^*$

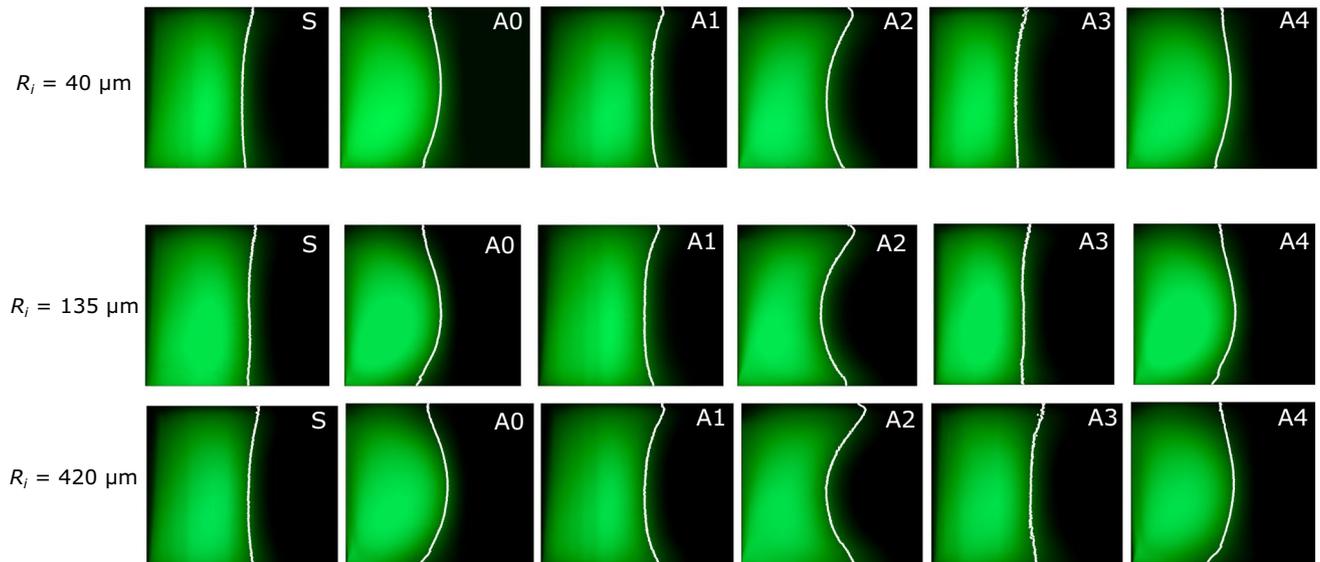
394 and the transverse (or wall normal) component is  $u_y$ . The  
395 data are plotted such that the zero  $y$  location corresponds  
396 to the inner edge at location A1. In Fig. 5a we plot the  
397 streamwise velocity component. First, we notice the good  
398 agreement between the experiments and the Newtonian  
399 simulation (black line and black symbols) where a slight  
400 asymmetry in the profile towards the inner wall is notice-  
401 able [as has been observed and discussed previously (Zilz  
402 et al. 2012)]. Second, the effect of elasticity on this main  
403 velocity component is rather subtle but, as is most eas-  
404 ily seen via inspection of the numerical profiles around  
405 the maximum of  $u_x$  (see inset of Fig. 5a), it is clear that  
406 elasticity acts to reduce this asymmetry by shifting the  
407 peak velocity back towards the centre of the channel. We  
408 then turn to the wall-normal velocity component, shown  
409 in Fig. 5b. For the Newtonian case this is essentially zero  
410 (within  $\pm 1\%$  of the bulk velocity)—in agreement with  
411 theoretical predictions for an inertialess duct of constant  
412 cross-sectional area and constant curvature (Lauga et al.  
413 2004). However, with the polymer solution, increasingly  
414 large wall-normal velocities (i.e. from the inner wall  
415 towards the outer) can be discerned as the flow rate (or  
416  $De^*$ ) is increased. At the highest  $De^*$  for both simulation  
417 and experiment these velocities reach  $\approx 0.15U$  at their  
418 peak. It can be observed that, much as is the case for  
419 the primary velocity component, these transverse veloc-  
420 ity profiles are also asymmetric with a peak closer to  
421 the inner wall. We believe that this is a consequence of  
422 the shear rate being larger close to the inner wall. As the  
423 streamwise normal stress—which, in combination with  
424 streamline curvature, is the driving force for the second-  
425 ary flow—increases with the shear rate, the higher shear  
426 rate at the inner wall leads to a concomitant asymmetry in  
427 the distribution of the secondary flow. Significant noise is  
428 visible on the experimental data, which is due to the dif-  
429 ficulty of resolving accurately a velocity component much  
430 smaller than the average velocity. The systematic small  
431 discrepancies between the numerically computed profiles  
432 and the experimental data may be caused by the uncer-  
433 tainty on  $Q_{\text{insta}}$  (known with a precision of  $\pm 1 \mu\text{l}/\text{min}$ ).

434 We quantify in Fig. 5c the increase in the magnitude  
435 of the secondary flow with the Deborah number  $De^*$ .  
436 The maximum of the numerically computed transverse  
437 velocity  $u_y$  scales linearly with  $De^*$  over the range of  
438 parameters considered, as expected far from the instabil-  
439 ity onset (Poole et al. 2013). The trend in the experimen-  
440 tal data is more difficult to resolve because of the noise  
441 level, which is particularly high compared to the expected  
442 velocities at the lower flow rates investigated. Qualita-  
443 tive agreement is nonetheless observed, with comparable  
444 magnitude for the secondary flow in both cases.

### 3.2 Cross-sectional visualisation of the flow using confocal microscopy

445  
446  
447 Except at the highest flow rates (or  $De^*$ ), the magnitude of  
448 the secondary flow velocities is very small, and thus difficult  
449 to resolve with PIV techniques. However, if the effect of  
450 these small velocities can be integrated over a large distance  
451 any effect should be magnified. One method to achieve this  
452 integrated effect is through the use of confocal microscopy  
453 in combination with dyed stream visualisation, which we  
454 now turn our attention to. For those experiments, the fluid  
455 supplied through one of the inlets is dyed with fluorescein.  
456 The location of the dyed stream is identified in Fig. 2. At the  
457 Y-junction, the two streams each occupy half of the channel  
458 width, separated by a straight centred interface in the plane  
459 of the channel cross-section. This interface is broadened by  
460 diffusion as the fluid travels downstream, and deformed in  
461 the region of the loops as the vortices of secondary flows  
462 transport fluid in the plane of the cross-section. Following  
463 the evolution of this interface by taking slices in the  $yz$  plane  
464 is thus a means of visualising the fluid transport that has  
465 occurred in the cross-sectional plane between consecutive  
466 slices. This evolution is shown in Fig. 6 for three channels  
467 with different inner radii of curvature, at the six locations  
468 identified in Fig. 2. The interface between the two streams  
469 of fluid is quite broad, due to molecular diffusion of the  
470 dye, but also due to the convolution of the image with a  
471 finite-sized point spread function, enhanced by the strong  
472 illumination conditions. Therefore, only the qualitative evo-  
473 lution of the interface can be obtained, by adjusting the light  
474 intensity at each  $z$  position with a diffusion-type profile. The  
475 inflection point of this profile provides an estimate of the  
476 location of the interface, which is marked by the bright lines  
477 in Fig. 6. This diffusion profile is wider towards the top and  
478 bottom, suggesting that Taylor dispersion is active in our  
479 system (Ismagilov et al. 2000). Coupled with the weaker  
480 light intensity close to the walls, this effect is responsible  
481 for a loss of resolution at the top and bottom wall, causing  
482 the slight bending of the interface observed in the straight  
483 channel (location S).

484 The top row in Fig. 6 shows the evolution of the interface  
485 in the channel we used for the  $\mu\text{PIV}$  measurements, at the  
486 largest  $De^*$  we investigated (0.86). This evolution is in good  
487 qualitative agreement with the numerically uncovered nature  
488 of the secondary flow as illustrated in Fig. 3a: between loca-  
489 tion S and A0, the fluid has travelled a quarter loop with the  
490 dyed stream at the inner edge of the bend. The convex shape  
491 of the interface at A0 indicates that the dye has been trans-  
492 ported towards the outer edge in the centreplane, and that  
493 un-dyed fluid has been carried towards the inner edge at the  
494 top and bottom walls. This is consistent with the transport  
495 expected from the numerically computed vortex structure.  
496 From A0 to A1, transport along a quarter loop of reversed



**Fig. 6** Top row: evolution of the cross-sectional view down the channel for the polymeric solution flown at  $De^* = 0.86$  in the channel used for the  $\mu$ PIV experiments: the white line highlights the position of the interface, as obtained by adjusting the horizontal diffusion profile of the dye. Middle and bottom rows: effect of the radius of curvature of the channel.  $Q = 14 \mu\text{l}/\text{min}$ ,  $R_i = 135 \mu\text{m}$  (middle row) and

$R_i = 420 \mu\text{m}$  (bottom row). Although the relative magnitude of the secondary flow is weaker than in the smaller channel, the larger  $Q$  and the integration over a longer distance make the vortical structure clear. The similarity of the interface evolution for both radii supports the scaling of the secondary flow with  $De$

497 curvature leads to the recovery of a straight interface, which  
 498 is bent to a concave shape after further transport in the same  
 499 half loop to location A2. The channel then reverses curvature  
 500 again, and a convex interface is observed after transport  
 501 over half a loop (location A4). The images at A0, A2 and  
 502 A4 are taken at the connection between consecutive half-  
 503 loops, where the channel reverses curvature. An additional  
 504 secondary flow is expected to be triggered from this sudden  
 505 change in curvature even in a Newtonian fluid (Guglielmini  
 506 et al. 2011). However, the displacement of the interface is  
 507 not sensitive to the local velocity field, but rather to the inte-  
 508 gration of the streamlines over the distance travelled in the  
 509 channel. Therefore, we expect that this is the reason we do  
 510 not seem to observe the signature of this secondary flow in  
 511 the cross-sectional profiles measured.

512 We will now use confocal visualisation to probe the  
 513 influence of the radius of curvature of the channel  $R_i$  on  
 514 the secondary flow, to gather further experimental proof  
 515 of the flow scaling with  $De$ . As the relative magnitude of  
 516 the secondary flow decreases for larger  $R_i$ , direct velocity  
 517 measurements are difficult for those geometries. However,  
 518 the integration over long distances makes the secondary  
 519 flow visible in confocal experiments. The second and  
 520 third rows in Fig. 6 show the evolution of the dyed and  
 521 un-dyed streams interface in two channels of larger radii  
 522 of curvature:  $R_i = 135 \mu\text{m}$  (middle row) and  $R_i = 420 \mu\text{m}$   
 523 (bottom row), but similar cross-section as the previous

channel.  $Q_{\text{insta}}$  depends non-linearly on  $R_i$ , therefore, for  
 524 those two larger channels we do not work in terms of quan-  
 525 tities scaled on the critical values, such as  $De^*$ . We keep  
 526 the flow rate constant, so that the ratio of the Deborah  
 527 numbers for both experiments is inversely proportional  
 528 to that of the channel radii:  $De_1/De_2 = R_{i,2}/R_{i,1}$ . Confo-  
 529 cal imaging of the cross-section shows clear evidence of  
 530 the cross-sectional vortices, with marked deflections of  
 531 the interface. The interface profiles obtained with the two  
 532 larger channels have similar curvature, which provides  
 533 semi-quantitative experimental evidence for the scaling  
 534 of  $u_y$  with  $De$ : the displacement of the interface between,  
 535 for instance, S and A0, is proportional to  $u_y \times \Delta t$ , where  
 536  $\Delta t$  is the time required for the base flow to travel from S  
 537 to A0.  $\Delta t \sim R_i/U \sim R_i/Q$  because the channels have the  
 538 same cross-section. Therefore, the lateral displacement of  
 539 the interface scales as  $u_y \times R_i/Q$ . As this displacement is  
 540 similar for the two channels, and  $Q$  is identical,  $u_y$  scales  
 541 as  $1/R_i$ , which is consistent with the linear scaling of  $u_y$   
 542 with  $De$  as measured numerically.  
 543

544 Finally, we also note that in all cases, the shape of the  
 545 interface is almost unchanged after transport over an even  
 546 number of consecutive half-loop (see A0 compared with  
 547 A4). We thus confirm experimentally for Deborah numbers  
 548 below one, memory effects are small in our system, as  
 549 predicted numerically (Poole et al. 2013).

## 4 Conclusions and outlook

The use of two complementary techniques,  $\mu$ PIV in the centreplane of the microfluidic device and confocal microscopy to image the cross-section of the device, has allowed us to perform one of the first experimental characterizations of steady, viscoelastic secondary flows in curved microchannels. The vortical structure of this flow in the cross-sectional plane, first unveiled by numerical calculations, was confirmed. Qualitative agreement is found in the flow profiles for the secondary transverse velocity. Those results improve our comprehension of viscoelastic flows in complex channel geometries, by validating the three-dimensional flow driven by the hoop stress in regions of constant curvature. A full understanding of the flow pattern in the serpentine channel, though, remains beyond the scope of our work: in the regions where the curvature is not constant (as is typically the case between consecutive half-loops), additional vortices may appear, which we do not discuss here. Their contribution to the flow dynamics in the serpentine microchannel may be important, though, via their interaction with the viscoelastic Dean flow we have characterised, and their position at a potentially critical location for the propagation of the elastic instability in the channel.

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